

## DETERMINATION OF CHARACTERISTIC FRACTURE MECHANICS VALUE AS RESISTANCE USING THE INSTRUMENTED IMPACT TEST

**Constantin STANESCU**, Professor Dr.eng. “Politechnica” of University Bucharest

e-mail: prof\_cstanescu@yahoo.com

**Adriana CARSTOIU**, Lecturer, Dr.eng. “Valachia” University of Targoviste

e-mail: adriana\_cirstoiu@yahoo.com\_

**Vasile SURCEL**, University of Targu Jiu

**Cristian BEBE**, eng Transport’s College of Bucharest

e-mail:cr\_bebe@yahoo.com

### ABSTRACT

The instrumented Charpy impact test is used for determining properties of impact strenght of plastics. It is as addition to the pendulum impact test according to ISO and is carried out on razor blade notched specimens. The range for the used plastics is from brittle therosets up to high impact polymer blends.Both, the load and the deflection signals are registered and the impact energy is divided into an elastic and a plastic part.Considering the requirements for the specimen size and notch geometry independent material parameters are calculated.These parameters can be used for control and protection of quality as well as for research and development.

### 1.Short description of the procedure

The testing of tough polymers is carried out by a pendulum Charpy impact tester PSW 4 with 4 J work capacity at maximum falling height. Using a Charpy impact device a single edge notched square specimen is broken by the pendulum hammer impact. The striker who has an edge on its impact side is fastened on a tubular pendulum arm. After releasing it moves in a circle and carries a part of its kinetic energy to the specimen at the lowest point of its trajectory.

The measuring device is able to register load(F)-time(t) diagrams and load(F)-deflection(f) diagrams, either. In the case of registered load-time diagrams first of all the deflection can be calculated following Newton’s 2<sup>nd</sup> axioma. In a first integration step (1) the velocity can be determined and in a second integration step (2) the deflection  $f$  of the specimen as a function of time.

$$v(t) = v_0 - \frac{1}{m} \int_0^t F(\tau) d\tau \quad (1)$$

$$f(t) = \int_0^t v(\tau) d\tau \quad (2)$$

An optical device is used for the measurement of the deflection. In the trace of the light rays a diaphragm fastened on the pendulum hammer is shifted by the motion of the

pendulum. Depending on the shifting more or less light rays reach a photo transistor. The resulting signal is amplified for further analysis.

Both, the load signal and the deflection signal are carried to a storage oscilloscope. The oscilloscope is used as an analogous digital converter and as a monitor for a first visual interpretation of the load-time curve and the load-deflection curve, replane of the pendulum on the opposite side of the striker edge (Fig. 1). The pendulum velocity should be 1.0 or 1.5 m/s.

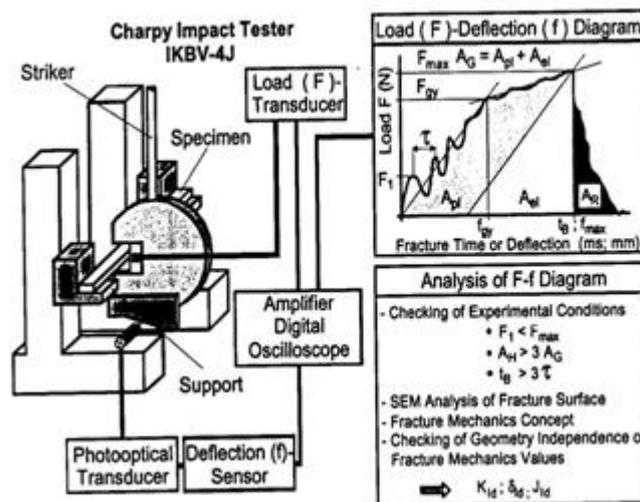


Fig. 1 Fracture mechanics testing device for the instrumented impact test

The registration of the load-deflection curves is carried out as described in the operating instructions.

The determination of the dynamic flexural modulus  $E_d$  and the dynamic yield stress  $\sigma_y$  is carried out on at least 5 unnotched specimens.

For that, the linear part of the load-deflection diagram is used, i.e., after fixing the yield point,  $E_d$  and  $\sigma_y$  can be calculated following (3) and (4), respectively.

$$E_d = \frac{F_{gy} s^3}{4BW^3 f_{gy}} \quad (3)$$

$$\sigma_y = \frac{3F_{gy} s}{2BW^2} \quad (4)$$

## 2. Determination of Fracture Mechanics Values

For the impact toughness evaluation of polymers under this procedure the following values are preferred:

Dynamic stress intensity factor  $K_{Qd}$

$$K_{Qd} = \frac{F_{max} s}{BW^{3/2}} f\left(\frac{a}{W}\right) \quad (5)$$

where

$$f\left(\frac{a}{W}\right) = \frac{3}{2} \left(\frac{a}{W}\right)^{1/2} \left[ \frac{1.99 - \frac{a}{W} \left(1 - \frac{a}{W}\right) \left(2.5 - 3.93 \frac{a}{W} + 2.7 \left(\frac{a}{W}\right)^2\right)}{\left(1 + 2 \frac{a}{W}\right) \left(1 - \frac{a}{W}\right)^{3/2}} \right] \quad (6)$$

and

$$f\left(\frac{a}{W}\right) = 2.9 \left(\frac{a}{W}\right)^{1/2} - 4.6 \left(\frac{a}{W}\right)^{3/2} + 21.8 \left(\frac{a}{W}\right)^{5/2} - 37.6 \left(\frac{a}{W}\right)^{7/2} + 38.7 \left(\frac{a}{W}\right)^{9/2} \quad (7)$$

respectively, is.

J-integral value  $J_{Qd}$

Evaluation method by Sumpter and Turner (ST)-  $J^{ST}$

$$J_{Qd}^{ST} = \eta_{el} \frac{A_{el}}{B(W-a)} + \eta_{pl} \frac{A_{pl}}{B(W-a)} \frac{W-a_{eff}}{W-a} \quad (8)$$

where

$$\eta_{el} = \frac{2F_{gy} s^2 (W-a)}{f_{gy} E_d B W^3} f^2 \left(\frac{a}{W}\right) (1-\nu^2) \quad (9)$$

and

$$\eta_{pl} = 2 - \frac{\left(1 - \frac{a}{W}\right) \left(0.892 - 4.476 \frac{a}{W}\right)}{1.125 + 0.892 \frac{a}{W} - 2.238 \left(\frac{a}{W}\right)^2} \quad (10)$$

Evaluation method by Merkle and Corten -  $J^{MC}$

$$J_{Qd}^{MC} = G_1 + \frac{2}{B(W-a)} [D_1 A_G + D_2 A_K - (D_1 + D_2) A_{el}] \quad (11)$$

$$G_1 = \frac{K_{ld}^2}{E} (1-\nu^2) \quad \text{for plane strain state} \quad (12)$$

$$D_1 = \frac{1 + \gamma}{1 + \gamma^2} \quad (13)$$

$$D_2 = \frac{\gamma(1 - 2\gamma - \gamma^2)}{(1 + \gamma^2)^2} \quad (14)$$

$$\gamma = \frac{1.456(W - a)}{s} \quad (15)$$

$$A_K = F_{\max} f_{\max} - A_G \quad (16)$$

Critical crack opening displacement  $\delta_{Qd}$

$$\delta_{Qd} = \frac{1}{n}(W - a) \frac{4f_{\max}}{s} \quad (17)$$

and

$$\delta_{Qdk} = \frac{1}{n}(W - a) \frac{4f_k}{s} \quad (18)$$

respectively, where

$$f_k = f_{\max} - f_b \quad (19)$$

The determination of further characteristic values results from technical requirements. These values must be reported in the testing protocol.

## REFERENCES:

- [1] Stanescu, C.D. , *Contributions to the analytical optimal synthesis of cames mechanism disk with follower oscillatory are employed the electronic computer HP*, In : Fifth World Congress on the Theory of machine and mechanism , vol.2,p 1287, Montreal, Canada, 1979;
- [2] Stanescu, C.D., *Theoretical and Tehnological contributions Kinematic of cam mechanism used automated machines-tools*, In: Ninth World Congress on the Theory of machine and mechanism, vol.1, p 709, Milan, Italy, 1995;
- [3] Stanescu, C. D., *Contributions a la synthese des mecanismes a comes utilisees aux machine outiles automatiques SARO 16-25*, In : Fifth world Congress on the Theory of machine and mechanism, vol.5, p 1415, Praga, Czech Republic, 1991;
- [4] Stanescu, C. D., *Theoretical research on operation accureacy of industrial robots*, In: Seventh International Congress on the Theory of machine and mecanism, p 571, Libereck, Czech Republic, 1996;
- [5] Stanescu, C.D., Voican Cristiana, *Theoretical Research on mechanisms used automated machines-tools*, In: Annual Scientific Papers, p 179, Oradea, 2001